

# Geometric Analysis of Rosette® Exit Cones

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An exploratory analysis of the geometry of Rosette® (registered trademark of Hitco) exit cones is carried out in the present work. The basic analysis begins with the premise that each horizontal cross section of the warped sheet (layer) follows a certain spiral trajectory governed by a set of equations which guarantee the continuity and axisymmetry requirements of the cone. While such a three-dimensional warped layer is not able to be developed from a flat basic sheet (e.g., a prepregged cloth material), a procedure based upon the preceding basic analysis is derived to construct the associated approximate flat sheet pattern. The errors associated with this procedure are calculated and found to be small for practical configurations, which indicates that the scheme is quite applicable in the actual fabrication of the rosette cone.

## Introduction

**E**XTENSIVE use of rosette construction in rocket nozzles and exit cones has led to an exploratory analytical study<sup>1</sup> in which the potential of rosette construction to produce striking reductions in peak stress concentration compared to hoop-wound and helical-wound cylinders was demonstrated. It is now appropriate to extend the study of Ref. 1 to solve a related problem of practical interest, namely, the Rosette® cone. The type of rosette construction treated here is defined by generating spirals which lie in planes normal to the axis of the cone. In the present work, our attention shall be confined to the geometric aspects of the problem. This leads to the definition of the basic sheet configuration from which the cone is constructed. While the detailed motion of the sheets in a rosette body is quite complex, this motion is immaterial for the present purpose, which is to describe the initial sheet pattern leading to the final geometry, which in turn is defined by the sheet thickness in the final configuration. The analysis leading to the description of mechanical response will be presented in a subsequent report.

It may be recalled that the basic sheet pattern in Ref. 1 was rectangular. These fibrous sheets (or layers) can be distorted and placed, layer by layer, along certain spiral trajectories until a cylindrical body is formed. In such an exact development process, the flexible flat sheets are transformed into a three-dimensional body with identical horizontal cross sections. For a rosette cone, however, the three-dimensional body is generated by a warped surface, since its horizontal spiral cross section varies both in length and curvature with respect to the cone axis (Fig. 1). Since such a surface is not able to be developed,<sup>2</sup> it becomes necessary to formulate an approximate treatment so that the conical structure can be built from prepregged cloth. To this end, a numerical mapping process will be carried out to determine the approximate configuration, to assess the associated error, and to

explore its sensitivity to the influence of the appropriate geometric parameters.

## Basic Geometry

Consider the frustrum of a cone illustrated in Fig. 1. The body is to be generated from a set of identical warped sheets of uniform thickness. Each surface of a warped sheet originates on the outer periphery of the cone (such as points along AB) and terminates on the inner circumference, e.g., curved line DC, or vice versa. Our present objective is to define the precise geometry of surface ABCD, which is consistent with the continuity requirement (complete filling of the volume with no gaps or overlaps). In the next section, the possibility of developing surface ABCD from a plane sheet shall be examined. The characteristic dimensions of the rosette cone are:

- $\alpha_0$  = arc angle at A, top point of a generator of the outer conical surface
- $R_0$  = outer radius of top ring
- $R_1$  = inner radius of top ring
- $R_2$  = inner radius of bottom ring
- $t$  = cone thickness
- $S$  = cone slant height along generator
- $\beta$  = cone half-angle
- $h$  = cone height

Guided by the results for the cylindrical case,<sup>1</sup> the analysis is begun by assuming that each warped sheet follows the trajectory

$$r \sin \alpha = c(z) \quad (1)$$

in every plane normal to the cone axis. Here  $c$  is a function of  $z$  alone. In Eq. (1),  $r$  is the radial distance in the cylindrical coordinate system and  $\alpha$  is the local arc angle. Also,  $\alpha_0$  is independent of  $\theta$ . In order to define the parameter  $c(z)$ , recall<sup>1</sup> that the number of layers  $N$  within a cylinder is given by

$$N = \frac{2\pi R_0 \sin \alpha_0}{t_l} = \frac{2\pi r \sin \alpha}{t_l} \quad (2)$$

where  $t_l$  is the thickness of a single layer measured in the plane  $z = \text{const}$ . Since  $N$  is a constant and  $t'$ , the ply thickness, and  $t_l$  are approximately related by  $t' = t_l \cos \beta$ , a good first approximation is given by

$$c(z) = \text{const} \quad (3)$$

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Index categories: Analytical and Numerical Methods; LV/M Structural Design (including Loads).

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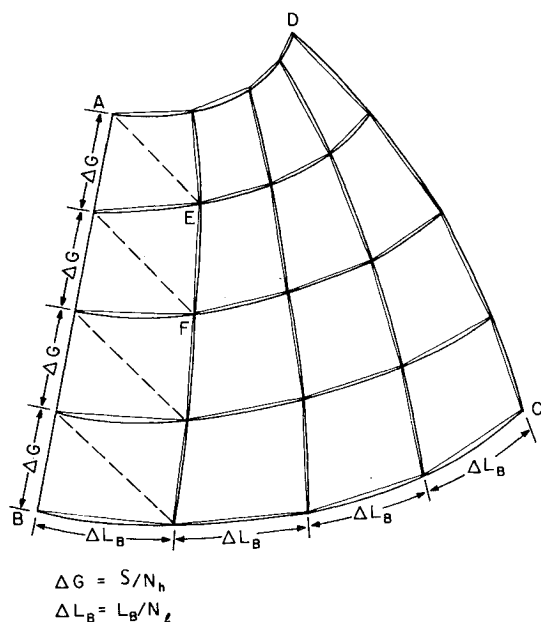


Fig. 3 Exact warped surface.

### Approximate Development

As mentioned earlier, the exact warped surface cannot be developed from a flat sheet. This is not surprising since warped surfaces are "those which may be generated by moving a straight line so that any two consecutive positions of the generating line are skew lines."<sup>2</sup> Consequently, it is necessary to adopt an approximate procedure. While classical approximate development approaches are available,<sup>2</sup> they depend upon discontinuous sheet patterns in the present case, i.e., cuts are required in the basic sheet pattern. Therefore, these methods are not considered acceptable. An alternate approach involves interconnection of a finite number of points on the surface to form a figure defined by a large number of triangular elements. As the elements shrink in size, this figure approaches the desired surface in the limit. While this figure can be developed exactly, the associated basic sheet pattern is again discontinuous. Another alternative involves derivation of a new sheet pattern, i.e., abandoning that given by Eq. (4). However, this approach destroys the axisymmetric feature and the continuity property of the geometry dictated by Eq. (4). Hence, it will be necessary to present a new approach to define an approximate development procedure for the warped basic sheet pattern. In this procedure, the lengths of line segments on the perimeter of the warped sheet are preserved in the limit.

The procedure begins by dividing the bottom spiral into  $N_z$  equally spaced points, as suggested by the intersection of the constant- $\theta$  and constant- $z$  trajectories. For fairly small values of  $N_z$ , however, the region adjacent to the right-hand border may be inadequately described in this approach. Therefore, a "boundary-layer" region is introduced here to avoid this lack of definition. This simply involves the use of additional constant- $\theta$  lines, those which originate at the intersection of the various  $z$  lines with the right-hand boundary. Next follows the computation of the true chord lengths joining the neighboring grid points. These are given by Eq. (13). At this time, the approximate process begins. As shown in Fig. 4, points 1 and 2 are first placed on the  $y$  axis at a distance  $\Delta G$  apart, where  $\Delta G = S/N_h$ . Point 1' is then determined by the intersection of chords  $d_1$  and  $d_2$ . This involves selecting the appropriate root from the solution of the simultaneous algebraic equations:

$$\begin{aligned} (x-x_1)^2 + (y-y_1)^2 &= d_1^2 \\ (x-x_2)^2 + (y-y_2)^2 &= d_2^2 \end{aligned} \quad (14)$$

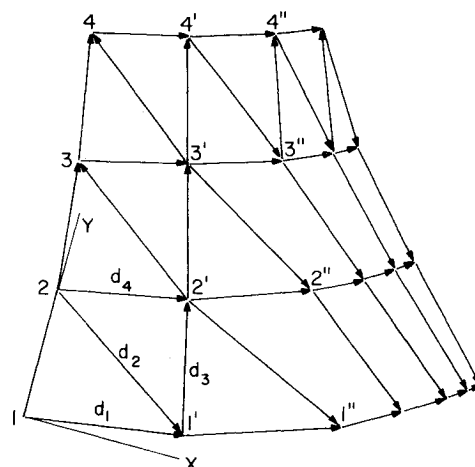


Fig. 4 Approximate development.

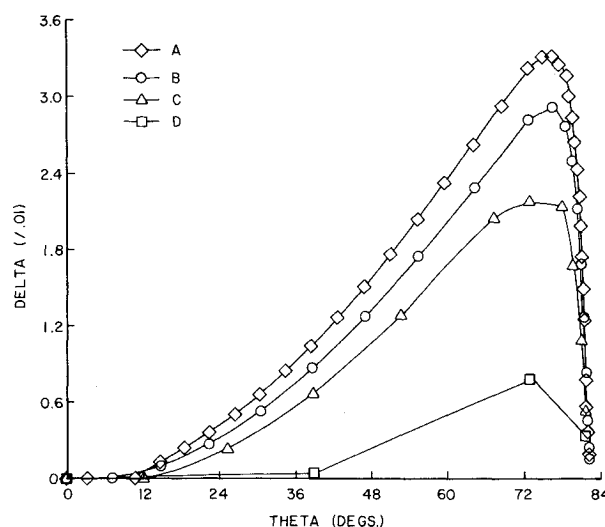


Fig. 5 Stretching displacement.

where  $(x_1, y_1)$  and  $(x_2, y_2)$  are the coordinates of the two known points. Likewise, point 2' can be determined by two chord lengths  $d_3$  and  $d_4$ , followed by points 3, 3', 4, 4', etc., until all the points on the first two  $\theta$  curves are mapped on this  $xy$  plane. For points corresponding to the remaining constant- $\theta$  curves, such as 1', 2', 3', etc., the same procedure has been taken, except that the true length requirement must be abandoned between 1'' and 2'', 2'' and 3'', i.e., between those points not connected in Fig. 4. This observation stems from consideration of the three-dimensional geometry of the triangular planes surrounding a point such as 2', 3', or 3''. Since these planes are, in general, distinct, the sum of the adjacent dihedral angles differs from 360 deg. This precludes a planar construction in which all triangles surrounding a given point are represented in true size. This is also an indication of the approximate nature of the present approach. The arrows represent true chords of the approximation process. It is also noted that the points on the right-hand boundary corresponding to the inner cone are located on different  $\theta$  lines. This is to be expected since the radial coordinates and the associated angles for these boundary points are

$$r_{\text{end}} = z \tan \beta - (R_0 - R_1) \quad (15)$$

and

$$\alpha_{\text{end}} = \sin^{-1} \left[ \frac{z \tan \beta - (R_0 - R_1)}{R_0 \sin \alpha_0} \right] \quad (16)$$

respectively. Therefore, via Eq. (5),  $\theta$  end depends on  $z$ .

One might expect the occurrence of numerical instabilities as the values of  $N_h$  and  $N_l$  become large. Such a problem, however, has not been encountered thus far with both the single- and double-precision options on the CDC 6600 system. In the event they do occur, a simple remedy involves omitting a number of  $\theta$  lines at a time along the right-hand boundary line. This procedure may also be applied to obtain a grid system with a more uniform appearance.

### Error Assessment

In the previous section, an approach was considered to perform an approximate development of the basic sheet element of a rosette cone. Here, a scheme is presented to define the quality of the approximation; i.e., to establish the credibility of the approximate development procedure. Conceptually, the scheme involves the assumption that the basic sheet may be modeled as a membrane, which is then stretched until the node points coincide with their counterparts on the exact surface. If the local values of the unit stretch are sufficiently small, the approximation is a good one. While it cannot be proven that the approach is universally valid, it can be shown, for configurations using exaggerated values of the input parameters (those which tend to magnify the errors and are greater than those used in practice), that the unit stretch distributions are quite small.

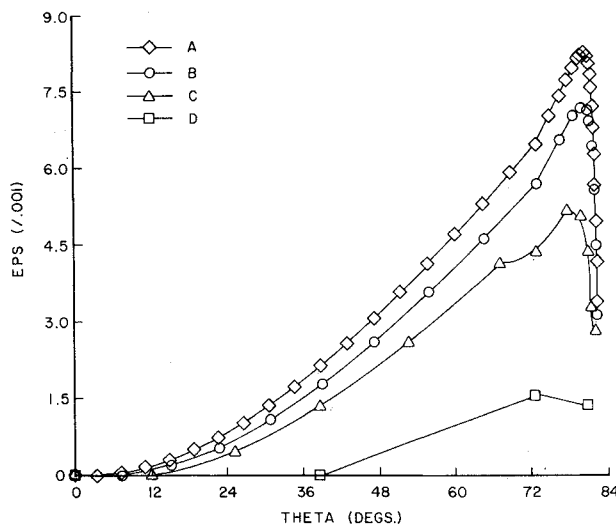


Fig. 6 Unit stretch (strain).

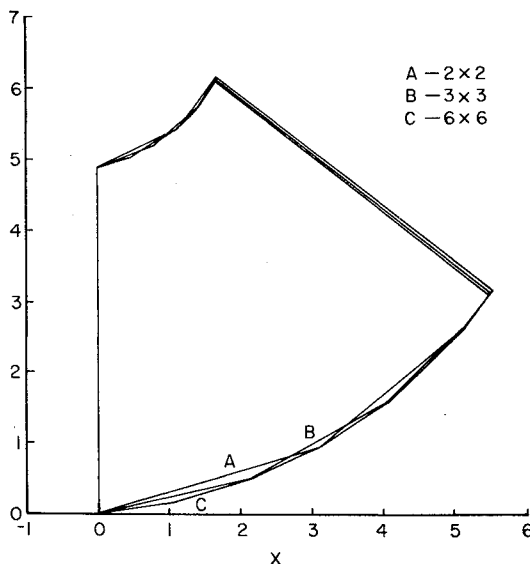


Fig. 7 Basic sheet pattern vs  $N_h = N_l$ .

Recall that true lengths of all the horizontal chords (Fig. 4) were employed throughout the approximate scheme. Clearly, as the value of  $N_l$  becomes large, the lengths of these chords will approach the corresponding segments of the spiral length. Hence, the unit stretch along the spirals approaches zero with an increase in  $N_l$ .

The unit stretch along the constant- $\theta$  lines, on the other hand, does not behave the same way. This can be observed from Figs. 5 and 6 (see Appendix for input data), where curves A, B, C, D, corresponding to  $N_h \times N_l = 20 \times 20$ ,  $10 \times 10$ ,  $6 \times 6$ , and  $2 \times 2$ , respectively, indicate that the errors are small. The  $\Delta$  (or DELTA) of Fig. 5 represents the difference between the constant- $\theta$  curve length and the respective length in the basic sheet pattern. The  $\epsilon$  (or EPS) of Fig. 6 is defined to be the ratio of  $\Delta$  to the  $\theta$  curve length. Note that all unit stretch values are several orders of magnitude smaller than unity, so that, for the given configuration, the approach is quite accurate. While it may appear the lower values of  $N_h$  and  $N_l$  lead to a better approximation, as indicated by Figs. 5 and 6, this is not necessarily true, since only the error due to stretching has been evaluated. Another measure of error exists; namely, the distance between a point on a node-connecting chord and its associated point on the exact surface, i.e., a measure of deflection. This deflection error, although quite large for small values of  $N_h$ , approaches zero as  $N_h$  becomes larger. Hence, the total error involves a combination of two effects; that is, the use of unit stretch alone is not a valid criterion for defining  $N_h$  and  $N_l$  to minimize the error in the approximate development. It is suggested, without proof, that larger values of  $N_h$  and  $N_l$  be used in practice. As these values approach infinity, examples have shown that a limiting, unique pattern is approached. For example, see Fig. 7 where patterns of  $2 \times 2$ ,  $3 \times 3$ , and  $6 \times 6$  are displayed as A, B, C, and Fig. 8, where the maximum unit stretch becomes asymptotic for large values of  $N$ .

### Concluding Remarks

A geometrical analysis of the Rosette® cone and an approximate approach to develop the basic sheet pattern have been presented. The basic analysis was based upon the rosette cylinder analysis of Ref. 1, where each horizontal cross

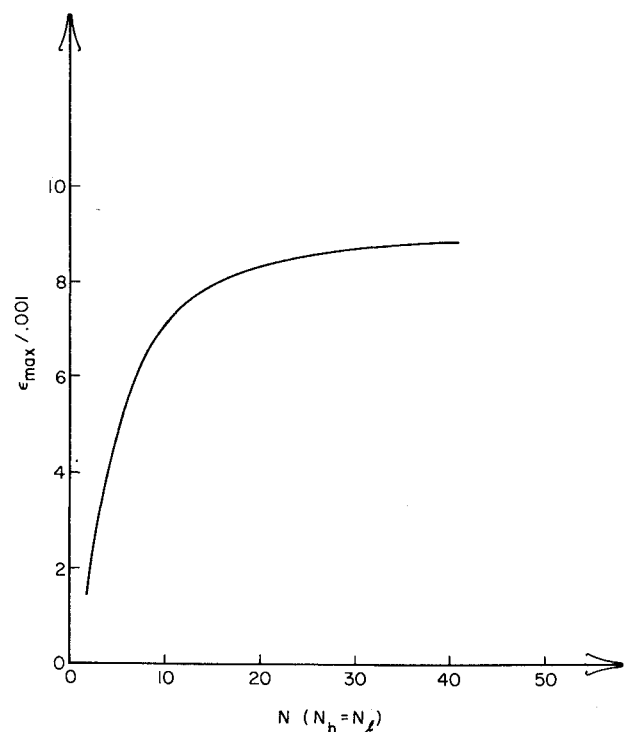


Fig. 8 Maximum unit stretch vs  $N$ .

section of the warped basic sheet follows a certain spiral trajectory dictated by Eqs. (5) and (6). While this warped surface is not developable from a flat sheet, results of the approximate approach indicated that the errors associated with the approximation are small for practical configurations. The concept of unit stretching was then introduced to deform the flat sheet until the node points coincide with their counterparts on the exact surface. From the resulting errors, it is concluded that the amount of stretching needed is small and the approximate scheme will be quite applicable in the actual fabrication of the rosette cone.

## Appendix

### Variable Wall Thickness

Consider a body in which the inner surface is that of a right circular cone and the outer surface is an arbitrary body of revolution. In this case, the warped sheet coincides with a generator along the inner surface. The relation

$$R_1 \sin \alpha_1 = R_0 \sin \alpha_0 \quad (A1)$$

defines the value  $\alpha_1$ , which is the starting value of  $\alpha$  in this case. Equation (A1) is employed to incorporate the appropriate parameters into Eqs. (2, 4, 5, and 6). Equations (8-10) are no longer valid, of course, since they were specifically developed for the case of constant wall thickness.

### Sample Input Data

The sample problem consists of the following input data:

$$\begin{aligned} \alpha_0 &= 20 \text{ deg} \\ R_0 &= 2 \text{ units} \\ R_1 &= 1 \text{ unit} \\ R_2 &= 4 \text{ units} \\ \beta &= \sin^{-1} [8/13] \\ h &= 4.875 \cos \beta \text{ units} \end{aligned} \quad (A2)$$

## Acknowledgments

The authors wish to express their appreciation to S.W. Tsai for his suggestions and discussion, to the UDRI Graphic Arts Section for preparing figures and vu-graphs, and to B. Drake and P. Thomas for typing the manuscript. They also wish to acknowledge the software assistance by J.S. Solomon, G. Fultz, D. Summers, and D. Walters.

## References

- <sup>1</sup>Pagano, N.J., "Elastic Response of Rosette Cylinders under Axisymmetric Loadings," *AIAA Journal*, Vol. 15, Feb. 1977, pp. 159-166.
- <sup>2</sup>Wellman, B.L., *Technical Descriptive Geometry*, McGraw-Hill Book Co., New York, 1948.
- <sup>3</sup>Conte, S.D. and deBoor, C., *Elementary Numerical Analysis*, McGraw-Hill Book Co., New York, 1972.
- <sup>4</sup>Abildskou, D.P.A., "Rosette Analysis," Hitco Stress Rept. AD-66-1, 1966.
- <sup>5</sup>Zak, A., "Rosette Flat Pattern Analysis," Hitco Stress Rept. AD-66-4, 1966.